Speed Reducers for Precision Motion Control

HarmonicDrive®
Reducer Catalog

- Differential Gear HDI
- Engineering Data

NOTE: This is a section from our full Reducer Product Catalog. Page numbers are for reference only.
Excellent Technology for Evolving Industries

Harmonic Drive® actuators utilize high-precision, zero-backlash Harmonic Drive® precision gears and play critical roles in robotics, semiconductor manufacturing equipment, factory automation equipment, medical diagnostics and surgical robotics. Additionally, our products are frequently used in mission-critical spaceflight applications which capture the human spirit.

With over 50 years of experience, our expert engineering and production teams continually develop enabling technologies for the evolving motion control market. We are proud of our outstanding engineering capabilities and successful history of providing customer specific solutions to meet their application requirements.

Harmonic Drive LLC continues to develop enabling technologies for the evolving motion control market, which drives the pace of global innovation.

C. Walton Musser
Patented Strain Wave
Gearing in 1955
Operating Principle of HarmonicDrive® Gears

A simple three-element construction combined with the unique operating principle puts extremely high reduction ratio capabilities into a very compact and lightweight package. The high-performance attributes of this gearing technology including, zero-backlash, high-torque-to-weight ratio, compact size, and excellent positional accuracy, are a direct result of the unique operating principles.

Harmonic Drive® gears have been evolving since the strain wave gear was first patented in 1955. Our innovative development and engineering teams have led us to significant advances in our gear technology. In 1988, Harmonic Drive successfully designed and manufactured a new tooth profile, the “S” tooth. Since implementing the “S” tooth profile, improvement in life, strength and torsional stiffness have been realized. In the 1990s, we focused engineering efforts on designing gears featuring space savings, higher speed, higher load capacity and higher reliability. Then in the 2000s, significant reduction in size and thickness were achieved, all while maintaining high precision specifications.
HDI Series

Infinite Indexer®

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NOTE: This is a section from our full Reducer Product Catalog. Page numbers are for reference only.
**Applications**

The Infinit-Indexer® phase adjuster provides the designer with a simple component which will solve an almost limitless variety of design problems through precise shaft phase adjustment.

**Features**

- Fine tune rotational position of shafts and machine parts
- Phase cams
- Adjust roll registration
- Take up backlash in spur and worm gears
- Synchronize indexing devices

**Differential gear HDI series**

Infinit-Indexer® phase adjusters are available from immediate stock in the standard bore sizes shown with keyways, set screws, and tapped holes for face mounting of either hub. It is possible for the user to modify these configurations by disassembling the unit. The hub material is easily machined low carbon steel. Available sizes are shown in the drawing below. Additional sizes and configurations are available by special order.

**HDI-25-8SK-8DK**

The rotary feed tables are driven by dual pinions which engage the ring gear. This unique design affords the ability to “take up” lost motion in the drive train and to actually preload the mechanism producing the “stiffness” necessary for rotary machining.

**Aligning Teeth of Gears or Sprockets**
**Infinit Indexer HDI**

**Factory Alignment Screws** are epoxy sealed - do not tamper or adjust.

**Adjusting Ring Locking Screw**

**Zero Backlash Wave Generator Adjusting Ring**

**Adjusting Ring Locking Screw**

**Table 313-1**

<table>
<thead>
<tr>
<th>Basic HDI Size</th>
<th>A</th>
<th>B</th>
<th>C</th>
<th>E</th>
<th>F</th>
<th>*Torque Rating lb.-in.</th>
<th>Approx. Weight lbs.</th>
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<td>1.06</td>
<td>.31</td>
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<td>-25</td>
<td>3.00</td>
<td>1.75</td>
<td>2.19</td>
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<td>-200</td>
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<td>2.38</td>
<td>.84</td>
<td>20,000</td>
<td>24.0</td>
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</table>

*Torque rating is for continuous one direction of rotation. For reversing torque systems, the tabulated rating is the sum of the CW & CCW torque.*

**Table 313-2**

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<th>Basic HDI Size</th>
<th>Bore Size</th>
<th>D Hub</th>
<th>S Hub</th>
<th>G</th>
<th>H</th>
<th>J</th>
<th>D Hub</th>
<th>S Hub</th>
<th>M* UN-2B</th>
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<td>2S</td>
<td>2DK</td>
<td>2SK</td>
<td>—</td>
<td>—</td>
<td>—</td>
<td>2DT</td>
<td>6-32</td>
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<td></td>
<td>.375</td>
<td>3D</td>
<td>3S</td>
<td>3DK</td>
<td>3SK</td>
<td>.409</td>
<td>.062</td>
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<td>3DT</td>
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<td>.561</td>
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<td>4DT</td>
<td>6-32</td>
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<td>6S</td>
<td>6DK</td>
<td>6SK</td>
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<td>.585</td>
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<td>.625</td>
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<td>5S</td>
<td>5DK</td>
<td>5SK</td>
<td>1/4-20</td>
<td>.710</td>
<td>.187</td>
<td>5ST</td>
<td>8-32</td>
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<td></td>
<td>.750</td>
<td>6D</td>
<td>6S</td>
<td>6DK</td>
<td>6SK</td>
<td>1/4-20</td>
<td>.837</td>
<td>.187</td>
<td>6ST</td>
<td>8-32</td>
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<tr>
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<td>.750</td>
<td>6D</td>
<td>6S</td>
<td>6DK</td>
<td>6SK</td>
<td>1/4-20</td>
<td>.863</td>
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<td>7SK</td>
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<td>.988</td>
<td>.250</td>
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<td>1.000</td>
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<td>8S</td>
<td>8DK</td>
<td>8SK</td>
<td>1/4-20</td>
<td>1.114</td>
<td>.250</td>
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<td>10-32</td>
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<td>8D</td>
<td>8S</td>
<td>8DK</td>
<td>8SK</td>
<td>1/4-20</td>
<td>1.114</td>
<td>.250</td>
<td>8ST</td>
<td>1/4-20</td>
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<tr>
<td></td>
<td>1.250</td>
<td>10D</td>
<td>10S</td>
<td>10DK</td>
<td>10SK</td>
<td>3/8-16</td>
<td>1.418</td>
<td>.375</td>
<td>10ST</td>
<td>5/16-18</td>
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<tr>
<td></td>
<td>1.625</td>
<td>13D</td>
<td>13S</td>
<td>13DK</td>
<td>13SK</td>
<td>3/8-16</td>
<td>1.793</td>
<td>.375</td>
<td>13ST</td>
<td>5/16-18</td>
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<tr>
<td></td>
<td>1.750</td>
<td>14D</td>
<td>14S</td>
<td>14DK</td>
<td>14SK</td>
<td>3/8-16</td>
<td>1.918</td>
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<td>14ST</td>
<td>5/16-18</td>
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<tr>
<td>-100</td>
<td>1.750</td>
<td>14D</td>
<td>14S</td>
<td>14DK</td>
<td>14SK</td>
<td>1/2-13</td>
<td>2.026</td>
<td>.625</td>
<td>14ST</td>
<td>3/8-16</td>
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<tr>
<td></td>
<td>2.000</td>
<td>16D</td>
<td>16S</td>
<td>16DK</td>
<td>16SK</td>
<td>1/2-13</td>
<td>2.276</td>
<td>.625</td>
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<td>3/8-16</td>
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<tr>
<td></td>
<td>2.500</td>
<td>20D</td>
<td>20S</td>
<td>20DK</td>
<td>20SK</td>
<td>1/2-13</td>
<td>2.778</td>
<td>.625</td>
<td>20ST</td>
<td>3/8-16</td>
</tr>
</tbody>
</table>

*Six holes equally spaced. True position .015 diameter except sizes -05 and -10 three holes. To order: specify the basic size and desired D and S hub configuration. Example: HDI 10-6D-6SK specifies a size -10 with D hub .750 diameter plain bore and S hub .750 diameter bore with keyway and setscrew.*
The Infinit-Indexer® phase adjuster can be installed in a machine system either as an in-line shaft coupling or a concentric shaft coupling.

In-Line Shaft (Fig. 1 & Fig. 2)

In order to properly align shafts concentric to one another, either the driven or driving shaft should pass completely through one hub and engage the other by an amount determined by the (L) dimension. The hubs are symmetrical; therefore, the (L) length applies to a piloting shaft length entering from either hub face.

The coupling is designed to transmit pure torque only. Radial reaction loads generated by gears, sprockets, shaft misalignment, etc., must be isolated from the unit by appropriate shaft bearing supports.

When it is not possible to maintain good shaft concentricity, it is recommended that the Infinit-Indexer be mounted in conjunction with a flexible coupling and adapter as shown in Fig. 2.

Concentric Shaft (Fig. 3)

The shaft should pass completely through the attached sprocket, gear, etc., and the Infinit-Indexer at a uniform diameter with a tight-running fit.

It is recommended that the region of the shaft under the gear, sprocket, etc., and connected hub be lightly lubricated with a multi-purpose grease at assembly.

**Adjusting Ring:**

One revolution of the knurled outer adjusting ring results in 3.6° of shaft phase adjustment. With the (D) hub fixed, rotation of the (S) hub is opposite to the direction of adjustment ring rotation. Conversely, with the (S) hub fixed, rotation at the (D) hub is in the same direction as adjusting ring rotation.

The coupling is essentially self-locking and applications requiring frequent adjustment can be investigated for the possibility of operating without having to seat the locking screw. However, those applications in which the coupling is subjected to typical motor start-up accelerations, sudden stops and/or a vibratory environment will require use of the screw to maintain a phase setting.

The coupling during adjustment is not intended to drive against any significant reaction load that may exist between the connected shafts. However, some adjusting ring torque amplification results to provide a hub drive torque capability within recommended limits noted to below:

**Lubrication:**

The unit is factory lubricated and will not require further maintenance under normal conditions. Nevertheless, periodic maintenance should be performed when unit is subject to frequent adjustment, dirty or other abnormal conditions, or when unit-adjusting torque becomes higher than normal.

<table>
<thead>
<tr>
<th>Unit Size</th>
<th>Adjusting Ring Torque (lb.)</th>
<th>Hub Drive Torque (lb. in.)</th>
</tr>
</thead>
<tbody>
<tr>
<td>-05</td>
<td>4</td>
<td>20</td>
</tr>
<tr>
<td>-10</td>
<td>8</td>
<td>40</td>
</tr>
<tr>
<td>-25</td>
<td>16</td>
<td>80</td>
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<td>-50</td>
<td>32</td>
<td>160</td>
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<td>-100</td>
<td>76</td>
<td>380</td>
</tr>
<tr>
<td>-200</td>
<td>150</td>
<td>750</td>
</tr>
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</table>

Spanner wrench holes are provided on the O.D. of the adjusting ring in sizes 50, 100, 200, and 300.

**Disassembly:**

Loosen friction adjustment/locking screw (it is not necessary to remove screw from unit) and remove one retaining ring. All parts will then slide out in one direction. (Do not tamper with or remove the two factory alignment screws.)

Clean parts and relubricate with multipurpose EP-2 grease.

**Reassembly:**

Assemble units with one ‘D’ and one ‘S’ hub (each is stamped). Unit will not phase adjust with two ‘S’ splines or two ‘D’ splines.

**Operation:**

Hand rotate the adjusting ring in either direction to produce a 100:1 reduction between the ring and one of the hubs. Adjust the friction adjustment/locking screw to desired resistance. For some applications, one adjustment will be sufficient for both shaft turning and phase adjusting modes. For more severe loading, such as hard stopping or higher torques, the friction adjustment/locking screw may be used to lock the adjusting ring in place to maintain phase.
If any two elements are locked together, the indexer will not phase and the unit will rotate in a 1:1 mode.

Precise manual displacement of roll centerline to adjust nip-roll pressure or depth-of-cut using HDI Infinit-Indexer® phase adjuster.

HDI for removal of backlash from a worm gear drive system. Two pinions, each mounted on the output shaft of separate, identical worm gear reducers, mate with a common bull gear. Adjusting the HDI causes one pinion to preload the bull gear against the other pinion. At setup, the assembler finds the loosest mesh point of the system and adjusts-out the backlash at that point. Any other position of the bull gear will result in a preloaded system.
Special Order
HDI phase adjusters are available in 6 sizes. All sizes are furnished complete with hubs to specific order requirements. Several bore sizes are available with keyways and tapped holes on one or both hubs or in minimum plain bore for alteration by the user.

Special Order by Model Ordering Code:

The Stocking Program
The stocking program offers the most cost effective way to purchase HDI phase adjusters. Three sizes of HDIs, (10, 25, and 50) are available from the stocking program.

Each comes with keyways and tapped holes on both hubs and is readily available from stock. Several bore sizes are available from the stocking program:

<table>
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<tr>
<th>HDI Size</th>
<th>Bore Sizes</th>
<th>Keyway</th>
<th>Tapped Holes</th>
<th>Torque Capacity</th>
<th>Model Ordering Code</th>
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<td>10</td>
<td>1/2&quot;</td>
<td>3/16&quot;</td>
<td>3 - #8-32</td>
<td>113 Nm</td>
<td>HDI - 010 - 500</td>
</tr>
<tr>
<td></td>
<td>5/8&quot;</td>
<td></td>
<td></td>
<td>1000 lb-in</td>
<td>HDI - 010 - 625</td>
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<tr>
<td></td>
<td>3/4&quot;</td>
<td></td>
<td></td>
<td></td>
<td>HDI - 010 - 750</td>
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<tr>
<td>25</td>
<td>3/4&quot;</td>
<td>1/4&quot;</td>
<td>6 - #10-32</td>
<td>283 Nm</td>
<td>HDI - 025 - 750</td>
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<td>1&quot;</td>
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<td></td>
<td>2500 lb-in</td>
<td>HDI - 025 - 10005</td>
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<td>50</td>
<td>1 1/4&quot;</td>
<td>1/4&quot;</td>
<td>6 - 1/4-20</td>
<td>565 Nm</td>
<td>HDI - 050-12500</td>
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Dimensions-Stocking Program

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<td>2.38</td>
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<td>1.69</td>
<td>1.69</td>
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<td>0.31</td>
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<td>1.06</td>
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Outline Dimensions

Figure 317-1

Outline Dimensions

Figure 317-1

Infinit Indexer HDI
Infinit Indexer HDI

Outline Dimensions

Figure 318-1

Infinit Indexer HDI

Outline Dimensions

Figure 318-1

Infinit Indexer HDI

Outline Dimensions

Figure 318-1

Infinit Indexer HDI

Outline Dimensions

Figure 318-1
MEMO
### Engineering Data

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<td>Design guideline</td>
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**Tooth Profile**

- **S tooth profile**

Harmonic Drive developed a unique gear tooth profile that optimizes the tooth engagement. It has a special curved surface unique to the S tooth profile that allows continuous contact with the tooth profile. It also alleviates the concentration of stress by widening the width of the tooth groove against the tooth thickness and enlarging the radius on the bottom. This tooth profile (the S tooth enables up to 30% of the total number of teeth to be engaged simultaneously).

Additionally the large tooth root radius increases the tooth strength compared with an involute tooth. This technological innovation results in high torque, high torsional stiffness, long life and smooth rotation.

*Patented*
Rotational direction and reduction ratio

Cup Style
Series: CSG, CSF, CSD, CSF-mini

Rotational direction

Input Output
(Note) Contact us if you use the product as an overdrive of (5) or (6).

(1) Reducer
Input: Wave Generator (WG)
Output: Flexspline (FS)
Fixed: Circular Spline (CS)
\[ i = \frac{1}{R} \]

(2) Reducer
Input: Wave Generator (WG)
Output: Circular Spline
Fixed: Flexspline
\[ i = \frac{1}{R+1} \]

(3) Reducer
Input: Flexspline
Output: Wave Generator
Fixed: Circular Spline
\[ i = \frac{R}{R+1} \]

(4) Overdrive
Input: Circular Spline
Output: Flexspline
Fixed: Wave Generator
\[ i = \frac{R+1}{R} \]

(5) Overdrive
Input: Flexspline
Output: Wave Generator (WG)
Fixed: Circular Spline
\[ i = \frac{1}{R} \]

(6) Overdrive
Input: Circular Spline
Output: Wave Generator (WG)
Fixed: Flexspline
\[ i = \frac{R}{R+1} \]

(7) Differential
When all of the wave generator, the flexspline and the circular spline rotate, combinations (1) through (6) are available.

Silk hat
Series: SHG, SHF, SHD

Rotational direction

Input Output
(Note) Contact us if you use the product as an overdrive of (5) or (6).

(1) Reducer
Input: Wave Generator (WG)
Output: Flexspline (FS)
Fixed: Circular Spline (CS)
\[ i = \frac{1}{R} \]

(2) Reducer
Input: Wave Generator (WG)
Output: Circular Spline
Fixed: Flexspline
\[ i = \frac{1}{R+1} \]

(3) Reducer
Input: Flexspline
Output: Wave Generator
Fixed: Circular Spline
\[ i = \frac{R}{R+1} \]

(4) Overdrive
Input: Circular Spline
Output: Flexspline
Fixed: Wave Generator
\[ i = \frac{R+1}{R} \]

(5) Overdrive
Input: Flexspline
Output: Wave Generator (WG)
Fixed: Circular Spline
\[ i = \frac{1}{R} \]

(6) Overdrive
Input: Circular Spline
Output: Wave Generator (WG)
Fixed: Flexspline
\[ i = \frac{R}{R+1} \]

(7) Differential
When all of the wave generator, the flexspline and the circular spline rotate, combinations (1) through (6) are available.
Pancake
Series: FB and FR

■ Rotational direction

1. Reducer
   - Input: Wave Generator
   - Output: Circular Spline
   - Fixed: Circular Spline
   - Reduction ratio: \( i = \frac{1}{R_1} = \frac{Z_c - Z_f}{Z_f} \)

2. Reducer
   - Input: Wave Generator
   - Output: Circular Spline
   - Fixed: Circular Spline
   - Reduction ratio: \( i = \frac{1}{R_2} = \frac{Z_c - Z_f}{Z_c} \)

3. Reducer
   - Input: Circular Spline D
   - Output: Circular Spline S
   - Fixed: Wave Generator
   - Reduction ratio: \( i = \frac{-R}{R + 1} \)

4. Overdrive
   - Input: Circular Spline S
   - Output: Wave Generator
   - Fixed: Circular Spline D
   - Reduction ratio: \( i = \frac{R + 1}{R} \)

5. Overdrive
   - Input: Circular Spline S
   - Output: Wave Generator
   - Fixed: Circular Spline D
   - Reduction ratio: \( i = \frac{R + 1}{R} \)

6. Overdrive
   - Input: Circular Spline D
   - Output: Wave Generator
   - Fixed: Circular Spline S
   - Reduction ratio: \( i = \frac{-R}{R + 1} \)

7. Differential
   - When all of the Wave Generator, the Circular Spline D rotates, Combinations (1) through (6) are available.

■ Reduction ratio

The reduction ratio is determined by the number of teeth of the Flexspline and the Circular Spline.

- Number of teeth of the Flexspline: \( Z_f \)
- Number of teeth of the Circular Spline: \( Z_c \)

\[ i = \frac{Z_c - Z_f}{Z_f} \]  
\[ i = \frac{Z_c - Z_f}{Z_c} \]

Example
- Number of teeth of the Flexspline: 200
- Number of teeth of the Circular Spline: 202

\[ i = \frac{200 - 202}{200} = \frac{-2}{100} \]

\[ i = \frac{202 - 202}{202} = \frac{0}{101} \]

* \( R_i \) indicates the reduction ratio value from the ratings table.
Rating Table Definitions

See the corresponding pages of each series for values.

■ Rated torque
Rated torque indicates allowable continuous load torque at rated input speed.

■ Limit for Repeated Peak Torque
(see Graph 12-1)
During acceleration and deceleration the Harmonic Drive® gear experiences a peak torque as a result of the moment of inertia of the output load. The table indicates the limit for repeated peak torque.

■ Limit for Average Torque
In cases where load torque and input speed vary, it is necessary to calculate an average value of load torque. The table indicates the limit for average torque. The average torque calculated must not exceed this limit. (calculation formula: Page 14)

■ Limit for Momentary Peak Torque
(see Graph 12-1)
The gear may be subjected to momentary peak torques in the event of a collision or emergency stop. The magnitude and frequency of occurrence of such peak torques must be kept to a minimum and they should, under no circumstance, occur during normal operating cycle. The allowable number of occurrences of the momentary peak torque may be calculated by using formula 13-1.

■ Maximum Average Input Speed
Maximum Input Speed
Do not exceed the allowable rating. (calculation formula of the average input speed: Page 14).

■ Moment of Inertia
The rating indicates the moment of inertia reflected to the gear input.

Life

■ Life of the wave generator
The life of a gear is determined by the life of the wave generator bearing. The life may be calculated by using the input speed and the output load torque.

<table>
<thead>
<tr>
<th>Series name</th>
<th>Life</th>
<th>Life of L10 or L50</th>
</tr>
</thead>
<tbody>
<tr>
<td>CSF, CSD, SHF, SHD, CSF-mini</td>
<td>7,000 hours</td>
<td>10,000 hours</td>
</tr>
<tr>
<td>CSG, SHG</td>
<td>35,000 hours</td>
<td>50,000 hours</td>
</tr>
</tbody>
</table>

Life is based on the input speed and output load torque from the rating table.

Calculation formula for Rated Lifetime
Formula 012-1

\[ L_h = L_n \left( \frac{T_r}{T_{a v}} \right) \cdot \left( \frac{N_r}{N_{a v}} \right) \]

<table>
<thead>
<tr>
<th>Ln</th>
<th>Life of L10 or L50</th>
</tr>
</thead>
<tbody>
<tr>
<td>Tr</td>
<td>Rated torque</td>
</tr>
<tr>
<td>Nr</td>
<td>Rated input speed</td>
</tr>
<tr>
<td>Tav</td>
<td>Average load torque on the output side (calculation formula: Page 14)</td>
</tr>
<tr>
<td>Nav</td>
<td>Average input speed (calculation formula: Page 14)</td>
</tr>
</tbody>
</table>

Example of application motion profile

Graph 012-1

Relative torque rating

Graph 012-2

* Life is based on the input speed and output load torque from the rating table.

* Lubricant life not taken into consideration in the graph described above.

* Use the graph above as reference values.
Torque Limits

■ Strength of flexspline
The Flexspline is subjected to repeated deflections, and its strength determines the torque capacity of the Harmonic Drive® gear. The values given for Rated Torque at Rated Speed and for the allowable Repeated Peak Torque are based on an infinite fatigue life for the Flexspline.
The torque that occurs during a collision must be below the momentary peak torque (impact torque). The maximum number of occurrences is given by the equation below.

Allowable limit of the bending cycles of the flexspline during rotation of the wave generator while the impact torque is applied: 1.0 x 10^4 (cycles)

The torque that occurs during a collision must be below the momentary peak torque (impact torque). The maximum number of occurrences is given by the equation below.

Calculation formula

\[ N = \frac{1.0 \times 10^4}{\frac{2}{60} \times n \times t} \]

**Caution**
If the number of occurrences is exceeded, the Flexspline may experience a fatigue failure.

■ Buckling torque
When a highly excessive torque (16 to 17 times rated torque) is applied to the output with the input stationary, the flexspline may experience plastic deformation. This is defined as buckling torque.

* See the corresponding pages of each series for buckling torque values.

**Warning**
When the flexspline buckles, early failure of the HarmonicDrive® gear will occur.

■ Ratcheting torque
When excessive torque (8 to 9 times rated torque) is applied while the gear is in motion, the teeth between the Circular Spline and Flexspline may not engage properly.
This phenomenon is called ratcheting and the torque at which this occurs is called ratcheting torque. Ratcheting may cause the Flexspline to become non-concentric with the Circular Spline.
Operating in this condition may result in shortened life and a Flexspline fatigue failure.

* See the corresponding pages of each series for ratcheting torque values.

* Ratcheting torque is affected by the stiffness of the housing to be used when installing the circular spline. Contact us for details of the ratcheting torque.

**Caution**
When ratcheting occurs, the teeth may not be correctly engaged and become out of alignment as shown in Figure 013-1. Operating the drive in this condition will cause vibration and damage the flexspline.

**Caution**
Once ratcheting occurs, the teeth wear excessively and the ratcheting torque may be lowered.

When excessive torque is applied, the teeth may not be correctly engaged and become out of alignment as shown in Figure 013-1. Operating the drive in this condition will cause vibration and damage the flexspline.

**Figure 013-1**

“Dedoidal” condition.

Once ratcheting occurs, the teeth wear excessively and the ratcheting torque may be lowered.

When excessive torque is applied, the teeth may not be correctly engaged and become out of alignment as shown in Figure 013-1. Operating the drive in this condition will cause vibration and damage the flexspline.

...
**Product Sizing & Selection**

In general, a servo system rarely operates at a continuous load and speed. The input rotational speed, load torque change and comparatively large torque are applied at start and stop. Unexpected impact torque may be applied. These fluctuating load torques should be converted to the average load torque when selecting a model number. As an accurate cross roller bearing is built in the direct external load support (output flange), the maximum moment load, life of the cross roller bearing and the static safety coefficient should also be checked.

![Graph 14-1](image)

**Flowchart for selecting a size**

Please use the flowchart shown below for selecting a size. Operating conditions must not exceed the performance ratings.

1. **Obtain the value of each application motion profile.**
   - **Load torque** \( T_n \) (Nm)
   - **Time** \( t \) (sec)
   - **Output rotational speed** \( n \) (rpm)

2. **Normal operation pattern**
   - **Starting (acceleration)** \( T_1, t_1, n_1 \)
   - **Steady operation** \( T_2, t_2, n_2 \)
   - **Stopping (deceleration)** \( T_3, t_3, n_3 \)
   - **Dwell** \( T_4, t_4, n_4 \)

3. **Maximum rotational speed**
   - **Max. output speed** \( n_{\text{max}} \)
   - **Max. input rotational speed** \( n_{\text{max}} \)
   - **(Restricted by motors)**

4. **Emergency stop torque**
   - When impact torque is applied \( T_s, t_s, n_s \)

5. **Required life**
   - \( L_{10} = L \) (hours)

6. **Calculate the average load torque applied on the output side from the application motion profile: \( T_{av} \) (Nm).**

\[
T_{av} = \frac{\sum n_i t_i T_i}{\sum n_i t_i}
\]

Make a preliminary model selection with the following conditions. \( T_{av} \leq \text{Limit for average torque treatment} \)

(See the rating table of each series).

7. **Calculate the average output speed: \( n_{av} \) (rpm)\)**

\[
n_{av} = n_i t_i T_i / \sum n_i t_i
\]

Obtain the reduction ratio \( R \).

A limit is placed on \( n_{\text{max}} \) by motors.

8. **Calculate the average input rotational speed from the max. output rotational speed (no \( n_{\text{max}} \)) and the reduction ratio \( R \): \( n_{\text{max}} \) (rpm).**

\[
n_{\text{max}} = \frac{n_{\text{max}}}{R}
\]

9. **Calculate (Ns) the allowable emergency stop torque.**

\[
N_s = \frac{10^4}{2 \times n_{\text{max}} \times R \times T_s}
\]

10. **Calculate the lifetime.**

\[
L_{10} = 7000 \left( \frac{T_{av}}{n_{av}} \right)^3 \left( \frac{\text{rpm}}{n_{\text{max}}} \right) \text{ (hours)}
\]

Check whether the calculated life is equal to or more than the life of the wave generator (see Page 13).

The model number is confirmed.
### Example of model number selection

#### Value of each application motion profile

<table>
<thead>
<tr>
<th>Load torque (Tav)</th>
<th>Time (t)</th>
<th>Output speed (n)</th>
</tr>
</thead>
<tbody>
<tr>
<td>Tav (Nm)</td>
<td>t (sec)</td>
<td>n (rpm)</td>
</tr>
</tbody>
</table>

#### Normal operation pattern

- **Starting (acceleration):** T1 = 400 Nm, t1 = 0.3 sec, n1 = 7 rpm
- **Steady operation (constant velocity):** T2 = 320 Nm, t2 = 3 sec, n2 = 14 rpm
- **Stopping (deceleration):** T3 = 200 Nm, t3 = 0.4 sec, n3 = 7 rpm
- **Dwell:** T4 = 0 Nm, t4 = 0.2 sec, n4 = 0 rpm

#### Maximum rotational speed

- Max. output speed: no max = 14 rpm
- Max. input speed: ni max = 1800 rpm

#### Emergency stop torque

When impact torque is applied: T5 = 500 Nm, t5 = 0.15 sec, n5 = 14 rpm

#### Required life

L10 = 7000 (hours)

#### Calculations

1. Calculate the average load torque to the output side based on the application motion profile: Tav (Nm).
   
   \[ Tav = 3 \times \frac{7 \text{ rpm} \times 0.3 \text{ sec} \times (400 \text{ Nm})^2 + 14 \text{ rpm} \times 3 \text{ sec} \times (320 \text{ Nm})^2 + 7 \text{ rpm} \times 0.4 \text{ sec} \times (200 \text{ Nm})^2}{7 \text{ rpm} \times 0.3 \text{ sec} + 14 \text{ rpm} \times 3 \text{ sec} + 7 \text{ rpm} \times 0.4 \text{ sec}} \]

2. Make a preliminary model selection with the following conditions. Tav = 319 Nm ≤ 620 Nm (Limit for average torque for model number CSF-40-120-2A-GR: See the rating table on Page 39.) Thus, CSF-40-120-2A-GR is tentatively selected.

3. Review the operation conditions, size and reduction ratio.

4. Check whether the preliminary selected model number satisfies the following condition from the rating table.
   
<table>
<thead>
<tr>
<th>Ni av</th>
<th>1440 rpm ≤ 3600 rpm (Max average input speed of size 40)</th>
<th>Ni max</th>
<th>1680 rpm ≤ 5600 rpm (Max input speed of size 40)</th>
</tr>
</thead>
<tbody>
<tr>
<td>OK</td>
<td>OK</td>
<td>NG</td>
<td>OK</td>
</tr>
</tbody>
</table>

5. Check whether T1 and T3 are equal to or less than the repeated peak torque specification.
   
   T1 = 400 Nm ≤ 617 Nm (Limit of repeated peak torque of size 40)
   T3 = 200 Nm ≤ 617 Nm (Limit of repeated peak torque of size 40)

6. Check whether T5 is equal to or less than the momentary peak torque specification.
   
   T5 = 500 Nm ≤ 1180 Nm (Limit for momentary torque of size 40)

7. Calculate the allowable number (Ns) rotation during impact torque and confirm \( 1.0 \times 10^4 \)
   
   \[ Ns = \frac{10^4}{2 \times \frac{14 \text{ rpm} \times 120}{60} \times 0.15 \text{ sec}} = 1190 \leq 1.0 \times 10^4 \]

8. Calculate the lifetime.
   
   \[ L_{10} = \frac{7000}{\frac{294 \text{ Nm}}{319 \text{ Nm}} \times \frac{2000 \text{ rpm}}{1440 \text{ rpm}}} \] (hours)

9. Check whether the calculated life is equal to or more than the life of the wave generator (see Page 12).
   
   L10 = 7610 hours ≥ 7000 (life of the wave generator: L10)

The selection of model number CSF-40-120-2A-GR is confirmed from the above calculations.
Lubrication

Component Sets: CSD-2A, CSF-2A, CSG-2A, F-2, F-0, FR-2, SHF-2A, SHG-2A and SHD and SHG/SHF -2SO and -2SH gear units: Grease lubricant and oil lubricant are available for lubricating the component sets and SHD gear unit. It is extremely important to properly grease your component sets and SHD gear unit. Proper lubrication is essential for high performance and reliability. Harmonic Drive® component sets are shipped with a rust- preventative oil. The characteristics of the lubricating grease and oil types approved by Harmonic Drive are not changed by mixing with the preservation oil. It is therefore not necessary to remove the preservation oil completely from the gear components. However, the mating surfaces must be degreased before the assembly.


Grease lubricant is standard for lubricating the gear units, you do not need to apply grease during assembly as the product is lubricated and shipped. See age 19 for using lubricant beyond the temperature range in table 16-2.

* Contact us if you want consistency zero (NLGI No.0) for maintenance reasons.

Grease lubricant

Types of lubricant

Harmonic Grease® SK-1A

This grease was developed for Harmonic Drive® gears and features good durability and efficiency.

Harmonic Grease® SK-2

This grease was developed for small sized Harmonic Drive® gears and features smooth rotation of the Wave Generator since high pressure additive is liquefied.

Harmonic Grease® 4B No.2

This has been developed exclusively for the CSF and CSG and features long life and can be used over a wide range of temperature.

(Note)

1. Grease lubrication must have proper sealing, this is essential for 4 No.2. Rotating part: Oil seal with spring is needed. Mating part: O ring or seal adhesive is needed.

2. The grease has the highest deterioration rate in the region where the grease is subjected to the greatest shear (near wave generator). Its viscosity is between JIS No.0 and No.00 depending on the operation.

<table>
<thead>
<tr>
<th>NLGI consistency No.</th>
<th>Mixing consistency range</th>
</tr>
</thead>
<tbody>
<tr>
<td>0</td>
<td>355 to 385</td>
</tr>
<tr>
<td>00</td>
<td>400 to 430</td>
</tr>
</tbody>
</table>

Grease specification

Name of lubricant

Table 016-1

<table>
<thead>
<tr>
<th>Grease</th>
<th>Table 016-1</th>
</tr>
</thead>
<tbody>
<tr>
<td>Harmonic Grease® S-1A</td>
<td></td>
</tr>
<tr>
<td>Harmonic Grease® S-2</td>
<td></td>
</tr>
<tr>
<td>Harmonic Grease® 4 No.2</td>
<td></td>
</tr>
<tr>
<td>Oil</td>
<td>Industrial gear oil class-2 (extreme pressure) ISO VG68</td>
</tr>
</tbody>
</table>

Temperature

Table 016-2

<table>
<thead>
<tr>
<th>Grease</th>
<th>Temperature</th>
</tr>
</thead>
<tbody>
<tr>
<td>S-1A</td>
<td>0 to + 40</td>
</tr>
<tr>
<td>S-2</td>
<td>0 to + 40</td>
</tr>
<tr>
<td>4 No.2</td>
<td>-10 to + 70</td>
</tr>
<tr>
<td>Oil</td>
<td>ISO VG68</td>
</tr>
<tr>
<td></td>
<td>0 to + 40</td>
</tr>
</tbody>
</table>

* The hottest section should not be more than 40° above the ambient temperature.

Note: The three basic components of the gear - the Flexspline, Wave Generator and Circular Spline - are matched and serialized in the factory. Depending on the product they are either greased or prepared with preservation oil. Then the individual components are assembled. If you receive several units, please be careful not to mix the matched components. This can be avoided by verifying that the serial numbers of the assembled gear components are identical.

Compatible grease by size

Compatible grease varies depending on the size and reduction ratio. See the following compatibility table. We recommend S-1A and S-2 for general use.

Ratios 30:1

Table 016-5

<table>
<thead>
<tr>
<th>Size</th>
<th>8</th>
<th>11</th>
<th>14</th>
<th>17</th>
<th>20</th>
<th>25</th>
<th>32</th>
</tr>
</thead>
<tbody>
<tr>
<td>S-1A</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
<tr>
<td>S-2</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
<tr>
<td>4 No.2</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
</tbody>
</table>

Ratios 50:1 and above

Table 016-6

<table>
<thead>
<tr>
<th>Size</th>
<th>40</th>
<th>45</th>
<th>50</th>
<th>58</th>
<th>65</th>
<th>80</th>
<th>90</th>
<th>100</th>
</tr>
</thead>
<tbody>
<tr>
<td>S-1A</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
<tr>
<td>S-2</td>
<td>△</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
<tr>
<td>4 No.2</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
</tbody>
</table>

Grease characteristics

Table 016-7

<table>
<thead>
<tr>
<th>Grease</th>
<th>SK-1A</th>
<th>SK-2</th>
<th>4B No.2</th>
</tr>
</thead>
<tbody>
<tr>
<td>Durability</td>
<td>○</td>
<td>○</td>
<td>○</td>
</tr>
<tr>
<td>Fretting resistance</td>
<td>△</td>
<td>-</td>
<td>-</td>
</tr>
<tr>
<td>Low-temperature performance</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
<tr>
<td>Grease leakage</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
<tr>
<td>Excellent</td>
<td>-</td>
<td>-</td>
<td>-</td>
</tr>
<tr>
<td>Good</td>
<td>△</td>
<td>○</td>
<td>○</td>
</tr>
</tbody>
</table>

* Oil lubrication is required for component-sets size 50 or larger with a reduction ratio of 50:1.

Grease characteristics
When to replace grease

The wear characteristics of the gear are strongly influenced by the condition of the grease lubrication. The condition of the grease is affected by the ambient temperature. The graph 017-1 shows the maximum number of input rotations for various temperatures. This graph applies to applications where the average load torque does not exceed the rated torque.

Note: Recommended Grease: S-1A or S-2

When to replace grease: \( L_{GTn} \) (when the average load torque is equal to or less than the rated torque)

\[
L_{GT} = L_{GTn} \times \left( \frac{Tr}{Tav} \right)^3
\]

Formula 017-1

Other precautions

1. Avoid mixing different kinds of grease. The gear should be in an individual case when installed.

2. Lease contact us when you use HarmonicDrive® gears at constant load or in one direction continuously, as it may cause lubrication problems.

3. Grease leakage. A sealed structure is needed to maintain the high durability of the gear and prevent grease leakage.

See the corresponding pages of the design guide of each series for recommended minimum housing clearance, application guide, and application quantity.
Precautions on using Harmonic Grease® 4B No.2

Harmonic Grease® 4B No.2 lubrication is ideally suited for Harmonic Drive® gears.

1. Apply the grease to each contacting joint at the beginning of operation.
2. Remove any contaminants created by abrasion during running-in period.

See the corresponding pages of the design guide of each series for “recommended minimum housing clearance,” Application guide” and “Application quantity.”

■ Precautions

1. Stir Grease
When storing Harmonic Grease 4B No.2 lubrication in the container, it is common for the oil to weep from the thickener. Before greasing, stir the grease in the container to mix and soften.

2. (2) Aging (running-in)
The aging before the main operation softens the applied grease. More effective greasing performance can be realized when the grease is distributed around each contact surface. Therefore, the following aging methods are recommended.
• Keep the internal temperature at 80°C or cooler. Do not start the aging at high temperature rapidly.
• Input rotational speed should be 1000rpm to 3000rpm. However, the lower rotational speed of 1000rpm is more effective. Set the speed as low as possible within the indicated range.
• The time required for aging is 20 minutes or longer.
• Operation range for aging: Keep the output rotational angle as large as possible.

Contact us if you have any questions for handling Harmonic Grease 4B No.2 lubrication.

Note: Strict sealing is required to prevent grease leakage.

Oil lubricant

■ Types of oil
The specified standard lubricant is “Industrial gear oil class-2 (extreme pressure) ISO VG68.” We recommend the following brands as a commercial lubricant.

<table>
<thead>
<tr>
<th>Standard</th>
<th>Mobil Oil</th>
<th>Exxon</th>
<th>Shell</th>
<th>COSMO Oil</th>
<th>Japan Energy</th>
<th>NIPPO Oil</th>
<th>Idemitsu Kosan</th>
<th>General Oil</th>
<th>Klüber</th>
</tr>
</thead>
<tbody>
<tr>
<td>Industrial gear oil class-2</td>
<td>Mobilgear 600XP68</td>
<td>Spartan EP68</td>
<td>Omaha Oil 68</td>
<td>Cosmo gear SE68</td>
<td>ES gear G68</td>
<td>Bonock M68, Bonock AX68</td>
<td>Daphne super gear LW68</td>
<td>General Oil SP gear roll 68</td>
<td>Syntheso D-68EP</td>
</tr>
<tr>
<td>(extreme pressure) ISO VG68</td>
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<td></td>
<td></td>
<td></td>
<td></td>
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<td></td>
</tr>
</tbody>
</table>

■ When to replace oil
First time ........................ 100 hours after starting operation
Second time or after .......... Every 1000 operation hours or every 6 months
Note that you should replace the oil earlier than specified if the operating condition is demanding.
See the corresponding pages of the design guide of each series for specific details.

■ Other precautions
1. Avoid mixing different kinds of oil. The gear should be in an individual case when installed.
2. When you use size 50 or above at max allowable input speed, please contact us as it may cause lubrication problems.

* Oil lubrication is required for component-sets size 50 or larger with a reduction ratio of 50:1.
When the ambient temperature is special (other than the “temperature range of the operating environment” on Page 016-2), you should select a lubricant appropriate for the operating temperature range.

Harmonic Grease 4B No.2

Table 019-1

<table>
<thead>
<tr>
<th>Type of lubricant</th>
<th>Operating temperature range</th>
<th>Available temperature range</th>
</tr>
</thead>
<tbody>
<tr>
<td>Grease</td>
<td>−10°C to + 110°C</td>
<td>−50°C to + 130°C</td>
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</tbody>
</table>

High temperature lubricant

Table 019-2

<table>
<thead>
<tr>
<th>Type of lubricant</th>
<th>Lubricant and manufacturer</th>
<th>Available temperature range</th>
</tr>
</thead>
<tbody>
<tr>
<td>Grease</td>
<td>Mobil grease 28: Mobil Oil</td>
<td>−5°C to + 160°C</td>
</tr>
<tr>
<td>Oil</td>
<td>Mobil SHC-626: Mobil Oil</td>
<td>−5°C to + 140°C</td>
</tr>
</tbody>
</table>

Low temperature lubricant

Table 019-3

<table>
<thead>
<tr>
<th>Type of lubricant</th>
<th>Lubricant and manufacturer</th>
<th>Available temperature range</th>
</tr>
</thead>
<tbody>
<tr>
<td>Grease</td>
<td>Multemp SH-KII: Kyodo Oil</td>
<td>−30°C to + 50°C</td>
</tr>
<tr>
<td></td>
<td>Isoflex LDS-18 special A: KLÜBER</td>
<td>−25°C to + 80°C</td>
</tr>
<tr>
<td>Oil</td>
<td>SH-200-100CS: Toray Silicon</td>
<td>−40°C to + 140°C</td>
</tr>
<tr>
<td></td>
<td>Syntheso D-32EP: KLÜBER</td>
<td>−25°C to + 90°C</td>
</tr>
</tbody>
</table>

Harmonic Grease 4B No.2

The operating temperature range of Harmonic Grease 4B No.2 lubrication is the temperature at the lubricating section with the performance and characteristics of the gear taken into consideration. (It is not ambient temperature.)

As the available temperature range indicates the temperature of the independent lubricant, restriction is added on operating conditions (such as load torque, rotational speed and operating cycle) of the gear. When the ambient temperature is very high or low, materials of the parts of the gear need to be reviewed for suitability. Contact us if operating in high temperature.

Harmonic Grease 4B No.2 can be used in the available temperature range shown in table 019-1. However, input running torque will increase at low temperatures, and grease life will be decreased at high temperatures due to oxidation and lubricant degradation.
Torsional Stiffness

Stiffness and backlash of the drive system greatly affects the performance of the servo system. Please perform a detailed review of these items before designing your equipment and selecting a model number.

■ Stiffness

Fixing the input side (wave generator) and applying torque to the output side (flex spline) generates torsion almost proportional to the torque on the output side. Figure 018-1 shows the torsional angle at the output side when the torque applied on the output side starts from zero, increases up to +T0 and decreases down to −T0. This is called the "Torque – torsion angle diagram," which normally draws a loop of 0 → A → B → A' → B' → A. The slope described in the "Torque – torsion angle diagram" is represented as the spring constant for the stiffness of the HarmonicDrive® gear (unit: Nm/rad).

As shown in Figure 020-1, this "Torque – torsional angle diagram" is divided into 3 regions, and the spring constants in the area are represented by K1, K2 and K3.

K1: The spring constant when the torque changes from [zero] to [T1]
K2: The spring constant when the torque changes from [T1] to [T2]
K3: The spring constant when the torque changes from [T2] to [T3]

■ See the corresponding pages of each series for values of the spring constants (K1, K2, K3) and the torque-torsional angles (T1, T2, - θ1, θ2).

■ Example for calculating the torsion angle

The torsion angle (θ) is calculated here using CSF-25-100-2A-GR as an example.

When the applied torque is T1 or less, the torsion angle θ1 is calculated as follows:
When the load torque T0 is 2.9 Nm
θ1 = T0/K1
= 2.9/3.1×10^3
= 9.4×10^-4 rad (0.33 arc min)

When the applied torque is between T1 and T2, the torsion angle θ2 is calculated as follows:
When the load torque is T2 is 39 Nm
θ2 = θ1 + (T2 − T1)/K2
= 4.4×10^-3 + (39−14)/5.0×10^3
= 9.4×10^-3 rad (0.33 arc min)

When a bidirectional load is applied, the total torsion angle will be 2 × θ plus hysteresis loss.

* The torsion angle calculation is for the gear component set only and does not include any torsional windup of the output shaft.

Note: See p.120 for torsional stiffness for pancake gearing.

■ Hysteresis loss (Silk hat and cup style only)

As shown in Figure 020-1, when the applied torque is increased to the rated torque and is brought back to [zero], the torsional angle does not return exactly back to the zero point This small difference (B − B') is called hysteresis loss.

■ See the corresponding page of each series for the hysteresis loss value.

Torque - torsion angle diagram

Spring constant diagram

■ Backlash (Silk hat and cup style only)

Hysteresis loss is primarily caused by internal friction. It is a very small value and will vary roughly in proportion to the applied load. Because HarmonicDrive® gears have zero backlash, the only true backlash is due to the clearance in the Oldham coupling, a self-aligning mechanism used on the wave generator. Since the Oldham coupling is used on the input, the backlash measured at the output is extremely small (arc-seconds) since it is divided by the gear reduction ratio.
Positional Accuracy

Positional Accuracy values represent the difference between the theoretical angle and the actual angle of output for any given input. The values shown in the table are maximum values.

See the corresponding pages of each series for transmission accuracy values.

Example of measurement

\[
\theta_{er} = \theta_1 - \frac{\theta_2}{R}
\]

Vibration

The primary frequency of the transmission error of the HarmonicDrive® gear may cause a vibration of the load inertia. This can occur when the driving frequency of the servo system including the HarmonicDrive® gear is at, or close to the resonant frequency of the system. Refer to the design guide of each series.

The primary component of the transmission error occurs twice per input revolution of the input. Therefore, the frequency generated by the transmission error is 2x the input frequency (rev / sec).

If the resonant frequency of the entire system, including the HarmonicDrive® gear, is \( f = 15 \text{ Hz} \), then the input speed (N) which would generate that frequency could be calculated with the formula below.

\[
N = \frac{15}{2} \cdot 60 = 450 \text{ rpm}
\]

The resonant frequency is generated at an input speed of 450 rpm.
Starting Torque

Starting torque is the torque value applied to the input side at which the output first starts to rotate. The values in the table of each series indicate the maximum value, and the lower-limit value indicates approximately \( \frac{1}{2} \) to \( \frac{1}{3} \) of the maximum value.

**Measurement conditions:**
No-load, ambient temperature: +20°C

- See the corresponding pages of each series for starting torque values.
- Use the values in the table of each series as reference values as they vary depending on the usage conditions.

Backdriving Torque

Backdriving torque is the torque value applied to the output side at which the input first starts to rotate. The values in the table are maximum values, typical values are approximately \( \frac{1}{2} \) of the maximum values.

Note: Never rely on these values as a margin in a system that must hold an external load. A brake must be used where back driving is not permissible.

**Measurement conditions:**
No-load, ambient temperature: +20°C

- See the corresponding pages of each series for backdriving torque values.
- Use the values in the table of each series as reference values as they vary depending on the usage conditions.
No-Load Running Torque

No-load running torque is the torque which is required to rotate the input side (high speed side), when there is no load on the output side (low speed side). The graph of the no-load running torque shown in this catalog depends on the measurement conditions shown in Table 023-1. Add the compensation values shown by each series to all reduction ratios except 100:1.

See the corresponding pages of each series for no-load running torque values.

Efficiency

The efficiency varies depending on the following conditions.

- Reduction ratio
- Input speed
- Load torque
- Temperature
- Lubrication (type and quantity)

The efficiency characteristics of each series shown in this catalog depends on the measurement condition shown in Table 023-2.

See the corresponding pages of each series for efficiency values.

Efficiency compensation coefficient

If load torque is below rated torque, a compensation factor must be employed. Calculate the compensation coefficient Ke from the efficiency compensation coefficient graph of each series and use the following example for calculation.

**Example of calculation**

Efficiency \( \eta \) (%) under the following condition is obtained from the example of CSF-20-80-2A-GR.

- Input rotational speed: 1000 rpm
- Load torque: 19.6 Nm
- Lubrication method: Grease lubrication (Harmonic Grease SK-1A)
- Lubricant temperature: 20°C

Since the rated torque of size 20 with a reduction ratio of 80 is 34 Nm (Ratings: Page 039), the torque ratio \( \alpha \) is 0.58. \( (\alpha=19.6/34=0.58) \)

- The efficiency compensation coefficient is \( Ke=0.93 \) from Graph 023-1.
- Efficiency \( \eta \) at load torque 19.6 Nm: \( \eta=Ke\cdot\eta_R=0.93 \times 78=73\% \)
Design Guidelines

Design guideline

The relative perpendicularity and concentricity of the three basic Harmonic Drive® elements have an important influence on accuracy and service life.

Misalignments will adversely affect performance and reliability. Compliance with recommended assembly tolerances is essential in order for the advantages of Harmonic Drive® gearing to be fully realized. Please consider the following when designing:

1. Input shaft, Circular Spline and housing must be concentric.

2. When operating, an axial force is generated on the wave generator. Input bearings must be selected to accommodate this axial load. See page 27.

3. Even though a HarmonicDrive® gear is compact, it transmits large torques. Therefore, assure that all required bolts are used to fastened the circular spline and flexspline and that they are tightened to the recommended torque.

4. As the flexspline is subject to elastic deformation, a minimal clearance between the flexspline and housing is required. Refer to “Minimum Housing Clearance” on the drawing dimension tables.

5. The input shaft and output shaft are supported by anti-friction bearings. As the wave generator and flexspline elements are meant to transmit pure torque only, the bearing arrangement needs to isolate the harmonic gearing from external forces applied to either shaft. A common bearing arrangement is depicted in the diagram.

6. A clamping plate is recommended (item 6). Its purpose is to spread fastening forces and to avoid any chance of making physical contact with the thin section of the flexspline diaphragm. The clamping plate shall not exceed the diaphragm’s boss diameter and is to be designed in accordance with catalog recommendations.

Fig. 024-1

Fig. 025-1
Design Guidelines

1. Input shaft, Circular Spline and housing must be concentric.

2. When operating, an axial force is generated on the wave generator. Input bearings must be selected to accommodate this axial load. See page 27.

3. Even though a HarmonicDrive® gear is compact, it transmits large torques. Therefore, assure that all required bolts are used to fastened the circular spline and flexspline and that they are tightened to the recommended torque.

4. As the flexspline is subject to elastic deformation, the A minimal clearance between the flexspline and housing is required. Refer to “Minimum Housing Clearance” on the drawing dimension tables.

5. The input shaft and output shaft are supported by anti-friction bearings. As the wave generator and flexspline elements are meant to transmit pure torque only, the bearing arrangement needs to isolate the harmonic gearing from external forces applied to either shaft. A common bearing arrangement is depicted in the diagram.

6. A clamping plate is recommended (item 6). Its purpose is to spread fastening forces and to avoid any chance of making physical contact with the thin section of the flexspline diaphragm. The clamping plate shall not exceed the diaphragm’s boss diameter and is to be designed in accordance with catalog recommendations.

The relative perpendicularity and concentricity of the three basic Harmonic Drive® elements have an important influence on accuracy and service life. Misalignments will adversely affect performance and reliability. Compliance with recommended assembly tolerances is essential in order for the advantages of Harmonic Drive® gearing to be fully realized. Please consider the following:

Fig. 024-1

For the component sets, both input and output shafts must be supported by two adequately spaced bearings in order to withstand external radial and axial forces without excessive deflection. In order to avoid damage to the component set when limited external loads are anticipated, both input and output shafts must be axially fixed.

Bearings must be selected whose radial play does not exceed ISO-standard C 2 class or “normal” class. The bearings should be axially and radially preloaded to eliminate backlash. Examples of correct bearing arrangements are shown in fig 025-1.
Wave generator

Structure of the wave generator
The wave generator includes an Oldham’s coupling type with a self-aligning structure and an integrated solid wave generator without a self-aligning structure, and which is used depends on the series.
See the diagram of each series for details. The basic structure of the wave generator and the shape are shown below.

![Diagram of Wave Generator](image1)

- Ball Separator
- Wave generator bearing
- Wave generator plug
- Insert
- Rubwasher
- Snap ring
- Wave generator hub

Structure of Oldham’s coupling
Fig. 026-2

![Diagram of Oldham’s Coupling](image2)

Axial Force of Wave Generator

<table>
<thead>
<tr>
<th>Size</th>
<th>F (N)</th>
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<tbody>
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<tr>
<td>7</td>
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<td>14</td>
<td>17</td>
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<td>80</td>
<td>90</td>
</tr>
<tr>
<td>90</td>
<td>100</td>
</tr>
</tbody>
</table>

Maximum axial force of the Wave Generator can be calculated by the formula shown below. The axial force may vary depending on its operating condition. The value of axial force tends to be a larger number when using high torque, extreme low speed and constant operation. The force is calculated (approximately) by the equation.

$$ F = \frac{V'}{\phi} \times \tan \theta $$

- $V'$: Max. hole diameter of wave generator
- $\phi$: Tapered holes are also available.
- $H$: Hole diameter of the wave generator hub with Oldham coupling
- $T$: Hole diameter of the wave generator hub
- $F$: Axial force

Maximum hole diameter of wave generator

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<tr>
<th>Size</th>
<th>Hole Diameter</th>
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<td>7</td>
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Output torque

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<th>Size</th>
<th>Torque (Nm)</th>
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<td>7</td>
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Table 027-1

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Table 027-2

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<th>Size</th>
<th>Torque (Nm)</th>
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<td>35000</td>
</tr>
<tr>
<td>100</td>
<td>40000</td>
</tr>
</tbody>
</table>

Calculation example

Output torque: 382 Nm
Reduction ratio: 50
Size: 32

- Axial force direction of the wave generator (maximum allowable momentary torque)
- Acceleration Axial force in direction for
- Deceleration Axial force in direction for

$$ F = \frac{382}{50} \times 0.07 \times \tan 30° $$

- Transmission torque
- Keyways should sustain the transmission torque.
- Maximum hole dimension. We recommend the dimension of keyway each size. The dimension can be changed within a range up to the standard hole dimension of the wave generator is shown for each series.
- Keyway includes the dimension of depth of keyway. It is necessary that the dimension of keyway is based on JIS standard. It is necessary that the dimension of keyway is based on JIS standard.
- Maximum hole dimension. We recommend the dimension of keyway each size. The dimension can be changed within a range up to the standard hole dimension of the wave generator is shown for each series.
**Maximum hole diameter of wave generator**

The standard hole dimension of the wave generator is shown for each size. The dimension can be changed within a range up to the maximum hole dimension. We recommend the dimension of keyway based on JIS standard. It is necessary that the dimension of keyways should sustain the transmission torque.

*Tapered holes are also available.*

In cases where a larger hole is required, use the wave generator without the Oldham coupling. The maximum diameter of the hole should be considered to prevent deformation of the Wave Generator plug by load torque. The dimension is shown in the table below and includes the dimension of depth of keyway.

(This is the value including the dimension of the depth of keyway.)

<table>
<thead>
<tr>
<th>Size</th>
<th>Standard dim. (H7)</th>
<th>Minimum hole dim.</th>
<th>Maximum hole dim.</th>
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</thead>
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<tr>
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</table>

**Hole diameter of the wave generator hub with Oldham coupling**

<table>
<thead>
<tr>
<th>Size</th>
<th>Max. hole dia. (V)</th>
<th>Min. plug thick (H)</th>
</tr>
</thead>
<tbody>
<tr>
<td>8</td>
<td>1.0</td>
<td>5.7</td>
</tr>
<tr>
<td>11</td>
<td>1.4</td>
<td>6.7</td>
</tr>
<tr>
<td>14</td>
<td>1.7</td>
<td>7.2</td>
</tr>
<tr>
<td>17</td>
<td>2.0</td>
<td>7.6</td>
</tr>
<tr>
<td>20</td>
<td>2.3</td>
<td>11.3</td>
</tr>
<tr>
<td>25</td>
<td>2.8</td>
<td>13.7</td>
</tr>
<tr>
<td>32</td>
<td>3.6</td>
<td>15.9</td>
</tr>
<tr>
<td>40</td>
<td>4.2</td>
<td>17.8</td>
</tr>
<tr>
<td>45</td>
<td>4.7</td>
<td>19.1</td>
</tr>
<tr>
<td>50</td>
<td>5.2</td>
<td>21.4</td>
</tr>
<tr>
<td>58</td>
<td>6.0</td>
<td>23.5</td>
</tr>
<tr>
<td>65</td>
<td>6.7</td>
<td>25.5</td>
</tr>
<tr>
<td>80</td>
<td>8.0</td>
<td>28.5</td>
</tr>
<tr>
<td>90</td>
<td>8.4</td>
<td>31.3</td>
</tr>
<tr>
<td>100</td>
<td>9.5</td>
<td>34.9</td>
</tr>
</tbody>
</table>

**Axial Force of Wave Generator**

When the gear is used to accelerate a load, the deflection of the Flexspline leads to an axial force acting on the Wave Generator. This axial force, which acts in the direction of the closed end of the Flexspline, must be supported by the bearings of the input shaft (motor shaft). When the gear is used to decelerate a load, an axial force acts to push the Wave Generator out of the Flexspline cup. Maximum axial force of the Wave Generator can be calculated by the equation shown below. The axial force may vary depending on its operating condition. The value of axial force tends to be a larger number when using high torque, extreme low speed and constant operation. The force is calculated (approximately) by the equation. In all cases, the Wave Generator must be axially (in both directions), as well as torsionally, fixed to the input shaft.

(Note)

Please contact us for further information on attaching the Wave Generator to the input (motor) shaft.

**Formula for Axial Force**

\[
F = \frac{382 \times \tan 30°}{(32 \times 0.00254)} \approx 380N
\]

**Symbols for Formula**

- **F**: Axial force
- **N**: Size
- **D**: Output torque
- **T**: m

**Calculation example**

Model name: CSF series
Size: 32
Reduction ratio: 50
Output torque: 382 Nm
(maximum allowable momentary torque)

\[
F = \frac{382}{(32 \times 0.00254)} \approx 380N
\]
### Assembly Precautions

#### Sealing
Sealing is needed to maintain the high durability of the gear and prevent grease leakage. Recommended for all mating surfaces, if the o-ring is not used. Flanges provided with o-ring grooves must be sealed when a proper seal cannot be achieved using the o-ring alone.

- **Rotating Parts**
  - Oil seal with spring is needed.
- **Mating flange**
  - O-ring or seal adhesive is needed.
- **Screw hole area**
  - Screws should have a thread lock (LOCTITE® 242 is recommended) or seal adhesive.

(Nota) If you use Harmonic Grease 4B No.2, strict sealing is required.

#### Assembly precautions
The wave generator is installed after the flexspline and circular spline. If the wave generator is not inserted into the flexspline last, gear teeth scuffing damage or improper eccentric gear mesh may result. Installation resulting in an eccentric tooth mesh (Dedoidal) will cause noise and vibration, and can lead to early failure of the gear. For proper function, the teeth of the flexspline and Circular Spline mesh symmetrically.

**■ Precautions on the wave generator**
1. Avoid applying undue axial force to the wave generator during installation. Rotating the wave generator bearing while inserting it is recommended and will ease the process.
2. If the wave generator does not have an Oldham coupling, extra care must be given to ensure that concentricity and inclination are within the specified limits.

**■ Precautions on the circular spline**
The circular Spline must not be deformed in any way during the assembly. It is particularly important that the mounting surfaces are prepared correctly.

1. Mounting surfaces need to have adequate flatness, smoothness, and no distortion.
2. Especially in the area of the screw holes, burrs or foreign matter should not be present.
3. Adequate relief in the housing corners is needed to prevent interference with the corner of the circular spline.
4. The circular spline should be rotatable within the housing. Be sure there is no interference and that it does not catch on anything.
5. When a bolt is inserted into a bolt hole during installation, make sure that the bolt fits securely and is not in an improper position or inclination.
6. Do not apply torque at recommended torque all at once. First, apply torque at about half of the recommended value to all bolts, then tighten at recommended torque. Order of tightening bolts must be diagonal.
7. Avoid pinning the circular spline if possible as it can reduce the rotational precision and smoothness of operation.

**■ Precautions on the flexspline**
1. Mounting surfaces need to have adequate flatness, smoothness, and no distortion.
2. Especially in the area of the screw holes, burrs or foreign matter should not be present.
3. Adequate clearance with the housing is needed to ensure no interference especially with the major axis of flexspline.
4. Bolts should rotate freely when installing through the mounting holes of the flexspline and should not have any irregularity due to the shaft bolt holes being misaligned or oblique.
5. Do not tighten the bolts with the specified torque all at once. Tighten the bolts temporarily with about half the specified torque, and then tighten them to the specified torque. Tighten them in an even, crisscross pattern.
6. The flexspline and circular spline are concentric after assembly. After installing the wave generator bearing, if it rotates in unbalanced way, check the mounting for dedoidal or non-concentric installation.
7. Care should be taken not to damage the flexspline diaphragm or gear teeth during assembly. Avoid hitting the tips of the flex spline teeth and circular spline teeth. Avoid installing the CS from the open side of the flexspline after the wave generator has been installed.

**■ Rust prevention**
Although the Harmonic Drive® gears come with some corrosion protection, the gear can rust if exposed to the environment. The gear external surfaces typically have only a temporary corrosion inhibitor and some oil applied. If an anti-rust product is needed, please contact us to review the options.

#### Sealing recommendations for gear units

<table>
<thead>
<tr>
<th>Area requiring sealing</th>
<th>Recommended sealing method</th>
</tr>
</thead>
<tbody>
<tr>
<td>Output side</td>
<td></td>
</tr>
<tr>
<td>Holes which penetrate housing</td>
<td>Use O-ring (supplied with the product)</td>
</tr>
<tr>
<td>Installation screw / bolt</td>
<td>Screw lock adhesive which has effective seal (LOCTITE® 242 is recommended)</td>
</tr>
<tr>
<td>Input side</td>
<td></td>
</tr>
<tr>
<td>Flange surfaces</td>
<td>Use O-ring (supplied with the product)</td>
</tr>
<tr>
<td>Motor output shaft</td>
<td>Please select a motor which has an oil seal on the output shaft.</td>
</tr>
</tbody>
</table>
“Dedoidal” state

It is normal for the flexspline to engage with the circular spline symmetrically as shown in Figure 029-1. However, if the ratcheting phenomenon, which is described on Page 013, is caused or if the three parts are forcibly inserted and assembled, engagement of the teeth may be out of alignment as shown in Figure 029-2. This is called “dedoidal”. Note: Early failure of the gear will occur.

■ How to check “dedoidal”

By performing the following methods, check whether the gear engagement is “dedoidal”.

1) Judging by the irregular torque generated when the wave generator turns
   1) Slowly turn the input shaft with your hand in a no-load condition. If you can turn it with average force, it is normal. If it turns irregularly, it may be “dedoidal”.
   2) Turn the wave generator in a no-load condition if it is attached to a motor. If the average current value of the motor is about 2 to 3 times the normal value, it may be “dedoidal”.

2) Judging by measuring vibration on the body of the flexspline

The scale deflection of the dial gauge draws a sine wave as shown by the solid line in Graph 029-3 when it is normally assembled. When “dedoidal” occurs, the gauge draws a deflected wave shown by the dotted line as the flexspline is out of alignment.
Checking Output Bearing

A precision cross roller bearing is built in the unit type and the gear head type to directly support the external load (output flange) (precision 4-point contact ball bearing for the CSF-mini series). Please calculate maximum moment load, life of cross roller bearing, and static safety factor to fully maximize the performance of a housed unit (gearhead).

- See the corresponding pages on each series for cross roller bearing specifications.

Checking procedure

1. **Checking the maximum moment load \((M_{\text{max}})\)**
   - Calculate maximum moment load \((M_{\text{max}})\).
   - Maximum moment load \((M_{\text{max}})\) \(\leq\) allowable moment \((M_c)\).

2. **Checking the life**
   - Calculate the radial load \((F_{\text{rav}})\) and the average axial load \((F_{\text{av}})\).
   - Calculate the radial load coefficient \((x)\) and the axial load coefficient \((y)\).
   - Calculate lifetime.

3. **Checking the static safety coefficient**
   - Calculate the static equivalent radial load coefficient \((o)\).
   - Check the static safety coefficient \((f_s)\).

How to calculate the maximum moment load

Maximum moment load \((M_{\text{max}})\) is obtained as follows. Make sure that \(M_{\text{max}} \leq M_c\).

\[
M_{\text{max}} = F_{\text{rav}} (L_r+R) + F_{\text{av}} \cdot L_a
\]

How to calculate the average load

When the radial load and axial load vary, the life of cross roller bearing can be determined by converting to an average load.

1. **How to calculate the average radial load \((F_{\text{rav}})\)**
   - Formula 031-1
   - Symbols for Formula 031-1
   - See Formula 031-1.

2. **How to calculate the average axial load \((F_{\text{av}})\)**
   - Formula 031-2
   - Symbols for Formula 031-2
   - See Formula 031-2.

3. **How to calculate the average output speed \((N_{\text{av}})\)**
   - Formula 031-3
   - Symbols for Formula 031-3
   - See Formula 031-3.

Symbols for Formula 030-1

<table>
<thead>
<tr>
<th>(F_{\text{rav}})</th>
<th>Max. radial load</th>
<th>N(\text{kgf})</th>
<th>See Fig. 030-1.</th>
</tr>
</thead>
<tbody>
<tr>
<td>(F_{\text{av}})</td>
<td>Max. axial load</td>
<td>N(\text{kgf})</td>
<td>See Fig. 030-1.</td>
</tr>
<tr>
<td>(L_r, L_a)</td>
<td>Offset amount</td>
<td>m</td>
<td>See Fig. 030-1 and specification of the output bearing of each series.</td>
</tr>
<tr>
<td>(R)</td>
<td>Offset amount</td>
<td>m</td>
<td>See Fig. 030-1 and specification of the output bearing of each series.</td>
</tr>
</tbody>
</table>
How to calculate the average load

(Average radial load, average axial load, average output speed)

When the radial load and axial load vary, the life of cross roller bearing can be determined by converting to an average load.

How to calculate the average radial load (Fr\text{av})

\[ Fr_{\text{av}} = \sqrt{\frac{n_1 t_1 (F_r t_1)^n + n_2 t_2 (F_r t_2)^n + \cdots + n_n t_n (F_r t_n)^n}{n_1 t_1 + n_2 t_2 + \cdots + n_n t_n}} \]

Note that the maximum radial load in t is Fr1 and the maximum radial load in t is Frn.

How to calculate the average axial load (Fa\text{av})

\[ Fa_{\text{av}} = \sqrt{\frac{n_1 t_1 (F_a t_1)^n + n_2 t_2 (F_a t_2)^n + \cdots + n_n t_n (F_a t_n)^n}{n_1 t_1 + n_2 t_2 + \cdots + n_n t_n}} \]

Note that the maximum axial load in t is Fa1 and the maximum axial load in t is Fan.

How to calculate the average output speed (Nav)

\[ Nav = \frac{n_1 t_1 + n_2 t_2 + \cdots + n_n t_n}{t_1 + t_2 + \cdots + t_n} \]

How to calculate the radial load coefficient (X) and axial load coefficient (Y)

\[ \frac{Fr_{\text{av}}}{Fr_{\text{av}+2} (L_r + P_a) + Fr_{\text{av}} \cdot L_a} = 1.5 \]
\[ \frac{Fr_{\text{av}}}{Fr_{\text{av}+2} (L_r + P_a) + Fr_{\text{av}} \cdot L_a} = 1.5 \]

Symbols for Formula 031-4

<table>
<thead>
<tr>
<th>Frav</th>
<th>Average radial load (N(kgf))</th>
</tr>
</thead>
<tbody>
<tr>
<td>Faav</td>
<td>Average axial load (N(kgf))</td>
</tr>
<tr>
<td>Lr, La</td>
<td>Offset amount (m)</td>
</tr>
<tr>
<td>R</td>
<td>Offset amount (m)</td>
</tr>
<tr>
<td>dp</td>
<td>Pitch circle diameter of a roller (m)</td>
</tr>
</tbody>
</table>
Engineering Data

**Life of the output bearing**

Calculate life of the output bearing by Formula 032-1.
You can calculate the dynamic equivalent radial load (Pc) by Formula 032-2.

### Formula 032-1

(Cross roller bearing)

\[ L_{10} = \frac{10^7}{60 \times N_{av}} \times \left( \frac{C}{f_w \cdot P_C} \right)^{0.3} \]

(4-point contact ball bearing)

\[ L_{10} = \frac{10^7}{60 \times N_{av}} \times \left( \frac{C}{f_w \cdot P_C} \right)^{0.3} \]

### Symbols for Formula 032-1

<table>
<thead>
<tr>
<th>Symbol</th>
<th>Description</th>
<th>Unit(s)</th>
</tr>
</thead>
<tbody>
<tr>
<td>L_{10}</td>
<td>Life</td>
<td>hour</td>
</tr>
<tr>
<td>N_{av}</td>
<td>Average output rated load speed</td>
<td>rpm</td>
</tr>
<tr>
<td>C</td>
<td>Basic dynamic rated load</td>
<td>N (kgf)</td>
</tr>
<tr>
<td>P_C</td>
<td>Dynamic equivalent</td>
<td>N (kgf)</td>
</tr>
<tr>
<td>f_w</td>
<td>Load coefficient</td>
<td>——</td>
</tr>
</tbody>
</table>

### Formula 032-2

\[ P_C = X \cdot \left( \frac{F_{av} + \frac{2(F_{av} \cdot L_r + R)}{dp}}{dp} \right) + Y \cdot F_{av} \]

### Symbols for Formula 032-2

<table>
<thead>
<tr>
<th>Symbol</th>
<th>Description</th>
<th>Unit(s)</th>
</tr>
</thead>
<tbody>
<tr>
<td>F_{av}</td>
<td>Average radial load</td>
<td>N (kgf)</td>
</tr>
<tr>
<td>F_{ax}</td>
<td>Average axial load</td>
<td>N (kgf)</td>
</tr>
<tr>
<td>dp</td>
<td>Pitch circle diameter</td>
<td>m</td>
</tr>
<tr>
<td>X</td>
<td>Radial load coefficient</td>
<td>——</td>
</tr>
<tr>
<td>Y</td>
<td>Axial load coefficient</td>
<td>——</td>
</tr>
<tr>
<td>L_r, L_a</td>
<td>———</td>
<td>m</td>
</tr>
<tr>
<td>R</td>
<td>Offset</td>
<td>m</td>
</tr>
<tr>
<td>M_{ave}</td>
<td>Average moment load</td>
<td>Nm</td>
</tr>
</tbody>
</table>

### Load coefficient

<table>
<thead>
<tr>
<th>Load status</th>
<th>f_w</th>
</tr>
</thead>
<tbody>
<tr>
<td>Steady operation without impact and vibration</td>
<td>1 to 1.2</td>
</tr>
<tr>
<td>Normal operation</td>
<td>1.2 to 1.5</td>
</tr>
<tr>
<td>Operation with impact and vibration</td>
<td>1.5 to 3</td>
</tr>
</tbody>
</table>

---

Note: A small angle of oscillation (less than 5 degrees) may cause fretting corrosion to occur since lubrication may not circulate properly. Contact us if this happens.
How to calculate life during oscillating motion

Calculate the life of the cross roller bearing during oscillating motion by Formula 033-1.

(Cross roller bearing)
\[
\text{Loc} = \frac{10^9}{60 \times n_1} \times \frac{90}{9} \times \left( \frac{C}{f_w \cdot P_c} \right)^{0.3}
\]

(4-point contact ball bearing)
\[
\text{Loc} = \frac{10^9}{60 \times n_1} \times \frac{90}{9} \times \left( \frac{C}{f_w \cdot P_c} \right)^{0.3}
\]

Symbols for Formula 033-1

<table>
<thead>
<tr>
<th>Symbol</th>
<th>Description</th>
<th>Unit</th>
</tr>
</thead>
<tbody>
<tr>
<td>Loc</td>
<td>Rated life for oscillating motion</td>
<td>hour</td>
</tr>
<tr>
<td>n_1</td>
<td>Round trip oscillation each minute</td>
<td>cpm</td>
</tr>
<tr>
<td>C</td>
<td>Basic dynamic rated load</td>
<td>N (kgf)</td>
</tr>
<tr>
<td>P_c</td>
<td>Dynamic equivalent radial load</td>
<td>N (kgf)</td>
</tr>
<tr>
<td>f_w</td>
<td>Load coefficient</td>
<td>——</td>
</tr>
<tr>
<td>(\theta)</td>
<td>Oscillating angle /2</td>
<td>Degree</td>
</tr>
</tbody>
</table>

(Note) A small angle of oscillation (less than 5 degrees) may cause fretting corrosion to occur since lubrication may not circulate properly. Contact us if this happens.
How to calculate the static safety coefficient

Basic static rated load is an allowable limit for static load, but its limit is determined by usage. In this case, static safety coefficient of the cross roller bearing can be calculated by Formula 034-2.

\[
 fs = \frac{Co}{Po}
\]

\[
 o = \frac{F_{max} + \frac{2M_{max}}{dp} + 0.44F_{a max}}{Co}
\]

<table>
<thead>
<tr>
<th>Operating condition of the roller bearing</th>
<th>fs</th>
</tr>
</thead>
<tbody>
<tr>
<td>When high rotation precision is required</td>
<td>≥2</td>
</tr>
<tr>
<td>When shock and vibration are expected</td>
<td>≥1.5</td>
</tr>
<tr>
<td>Under normal operating condition</td>
<td>≥1.5</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>Symbols for Formula 034-1</th>
<th>Symbols for Formula 034-2</th>
</tr>
</thead>
<tbody>
<tr>
<td>Co</td>
<td>Co</td>
</tr>
<tr>
<td>Po</td>
<td>Po</td>
</tr>
<tr>
<td>Basic static rated load N(kgf)</td>
<td>See &quot;Specification of the output bearing&quot; of each series.</td>
</tr>
<tr>
<td>Static equivalent radial load N(kgf)</td>
<td>See Formula 034-2.</td>
</tr>
<tr>
<td>F_{max} Max. radial load N(kgf)</td>
<td>F_{max} Max. radial load N(kgf)</td>
</tr>
<tr>
<td>F_{a max} Max. axial load N(kgf)</td>
<td>See &quot;Specification of the output bearing&quot; of each series.</td>
</tr>
<tr>
<td>M_{max} Max. moment load Nm(kgf/m)</td>
<td>See &quot;Specification of the output bearing&quot; of each series.</td>
</tr>
<tr>
<td>dp Pitch circle diameter of a roller m</td>
<td>dp Pitch circle diameter of a roller m</td>
</tr>
</tbody>
</table>

When high rotation precision is required

When shock and vibration are expected

Under normal operating condition

See "Specification of the output bearing" of each series.
HarmonicDrive® Reducer for Precision Control

For Safe Use of Harmonic Drive® Component Set and Gear Unit

⚠️ Warning ⚠️: Means that improper use or handling could result in a risk of death or serious injury.

⚠️ Caution ⚠️: Means that improper use or handling could result in personal injury or damage to property.

Limited Applications

This product cannot be used for the following applications:
- Space equipment
- Aircraft equipment
- Nuclear power equipment
- Equipment and apparatus used in domestic homes
- Equipment that directly works on human bodies
- Equipment for transport of humans
- Equipment for use in a special environment

Please consult Harmonic Drive beforehand when intending to use one of its product for the aforementioned applications.

Install a safety device that avoids an accident even if output of this product becomes uncontrollable due to breakdown when using it in equipment that affects human lives and that may trigger serious damage.

Design Precaution: Be certain to read the catalog when designing your equipment.

- Use only in a specified environment.
  - Please ensure the following environmental conditions are complied with:
    - Ambient temperature 0 to 40°C
    - No splashing of water or oil
    - No dust such as metal powder
- Install the equipment in a specified manner.
  - Carry out assembly precisely in the specified order according to the catalog.
  - Observe our recommended tightening methods (such as bolts used).
  - Operating the equipment without precise assembly can cause troubles such as generation of vibration, reduction of life, deterioration of precision and breakdown.

Operational Precaution: Be certain to read the catalog before operating the equipment.

- Be careful in handling products and parts.
  - Do not impact the parts and units with a hammer, etc. Do not scratch or dent them. Possible damage is assumed.
  - If you use the equipment in a damaged condition, the specified performance may not be retained. It can also cause troubles such as breakdown.
- Apply torque within the allowable range.
  - Do not apply torque exceeding the instantaneous allowable max. torque. Applying excess torque can cause troubles such as loose tightening bolts, generation of backlash and breakdown.
  - Sinking an arm directly attached to the output shaft can damage the arm and make the output shaft uncontrollable.
- Do not change a set of parts.
  - The product is manufactured with matched serial numbers. The specified performance may not be retained if you have used mixed sets of parts.
- Do not break down unit products.
  - Do not break down and reassemble unit products. Original performance may not be reproduced.

Handling lubricant

- Precautions on handling lubricant
  - Use protective eyewear when handling. Avoid contact with eyes. May cause inflammation.
  - Use gloves and other skin protection when handling. Avoid contact with skin. May cause inflammation.
  - Do not swallow. May cause diarrhea and vomiting.
  - Keep out of reach of children.
- Treatment of waste oil and containers
  - Proper disposal is essential. Dispose of waste oil and waste containers properly in accordance with applicable laws and regulations.
  - Do not apply pressure on an empty container.
  - Do not weld, heat, drill or cut the container. The remainder may ignite with an explosion.

First-aid

- EYES: Flush with clean water for at least 15 minutes. Consult a physician.
- SKIN: Thoroughly remove with cloth or paper and wash with soap and water.
- NGESTION: Consult a physician immediately. Do not induce vomiting.
- INHALATION: Immediately seek fresh air if adverse effects are observed.
- Storage
  - Close the container tightly after use in order to prevent contamination with dust or moisture. Avoid direct exposure to sunlight. Store at room temperature.

When Discarding Actuator and Servo Driver

- Please discard as industrial waste.